



The European Commission's
science and knowledge service

Joint Research Centre

Directorate E

Space, Security and Migration

Unit E.4

Safety and Security of Buildings

A MATLAB Toolbox for Experimental Modal Analysis of Structures

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Milano

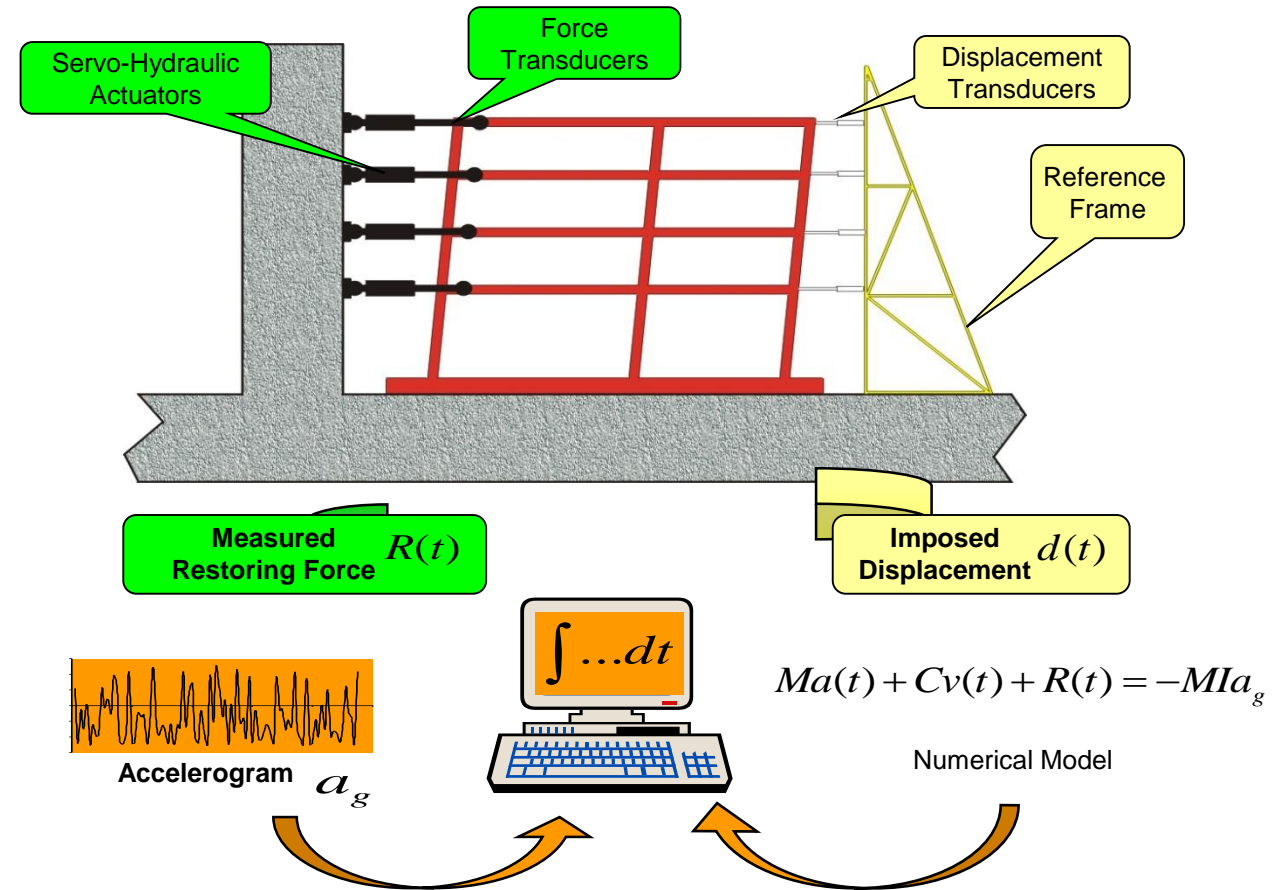




ELSA Laboratory: The European Laboratory for Structural Assessment



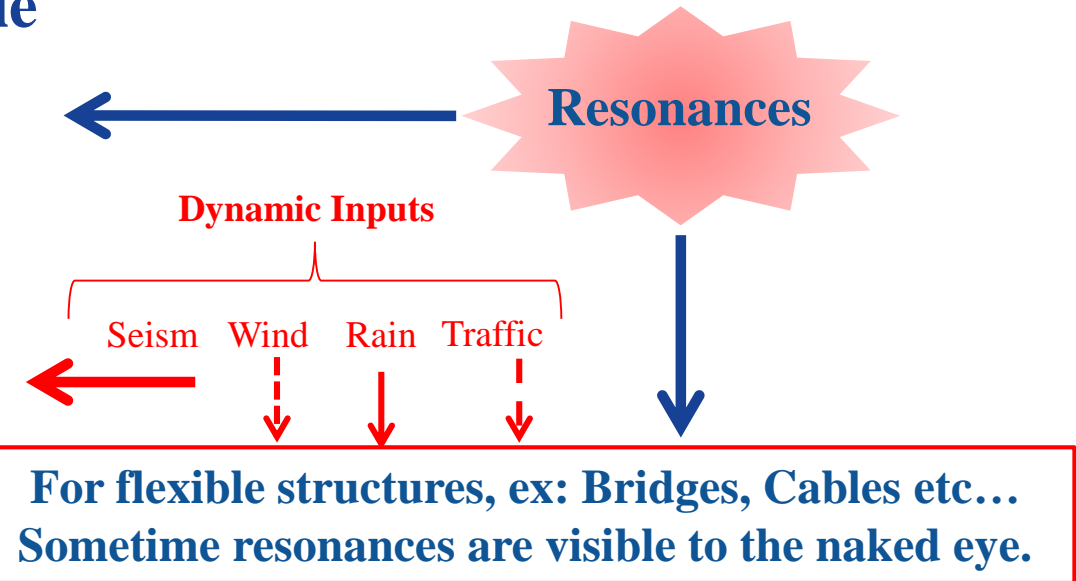
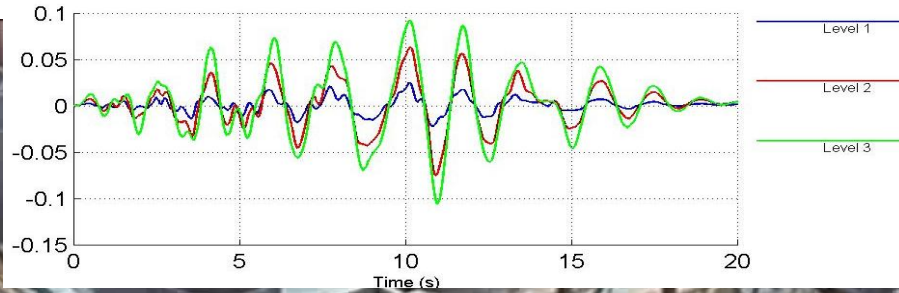
Activity mainly for European Standards
in Civil Engineering



The Pseudo Dynamic Method



Example of bi-directional test at real scale

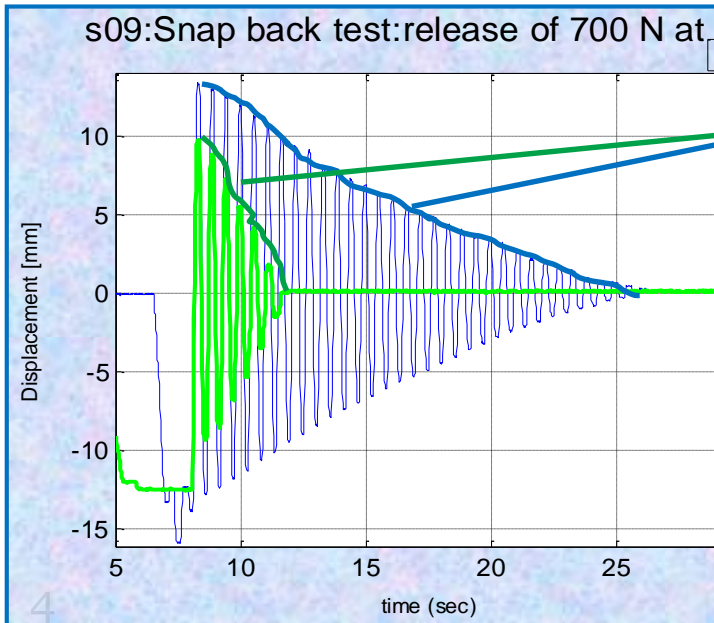
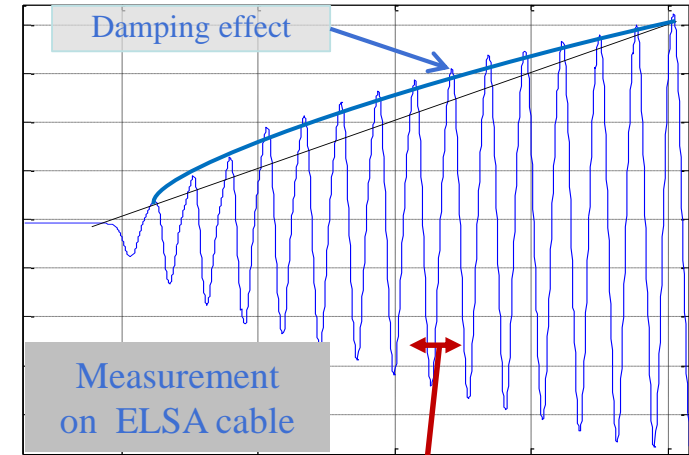


Storm on Volgograd Bridge (Russia 2010). Source: *YouTube*



What to understand from resonances?

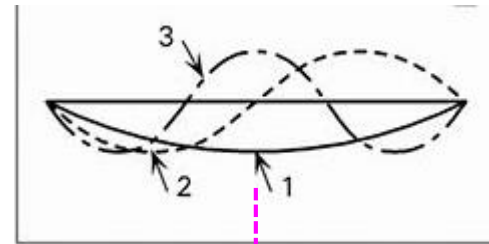
Dangerous for structures



Damping ζ

Mode Shapes ϕ_i Amplitude at different positions

Frequency



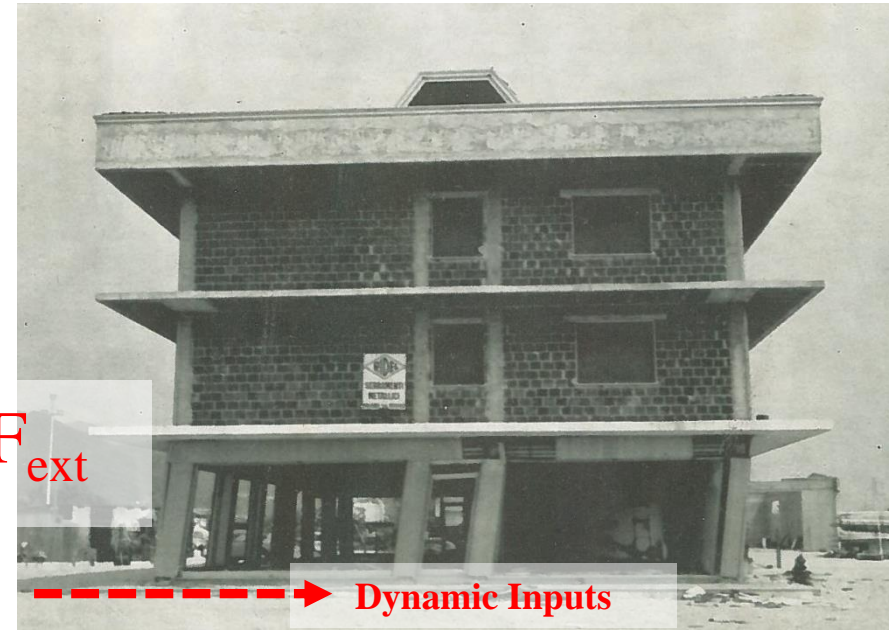
From measurements at different positions of the structure the toolbox calculates the three parameters which govern resonances

They depend from the structure only (not from the input).

Equation of motion of the structure

Inertial Forces Damping Forces Reaction Forces

$$[M]\{x''\} + [2.\zeta.\omega]\{x'\} + [K]\{x\} = F_{ext}$$



Source: Messaggero Veneto. Friuli Earthquake 1976

The three parameters calculated by the toolbox from experiments

Frequency ~ $(K_i/M_i)^{1/2}$

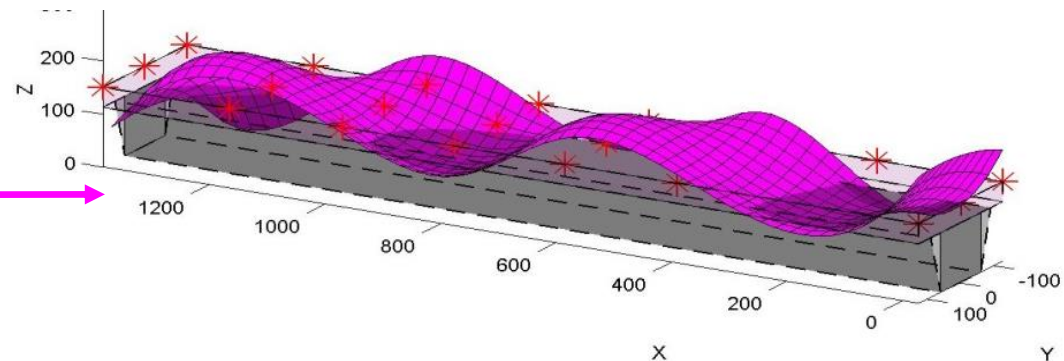
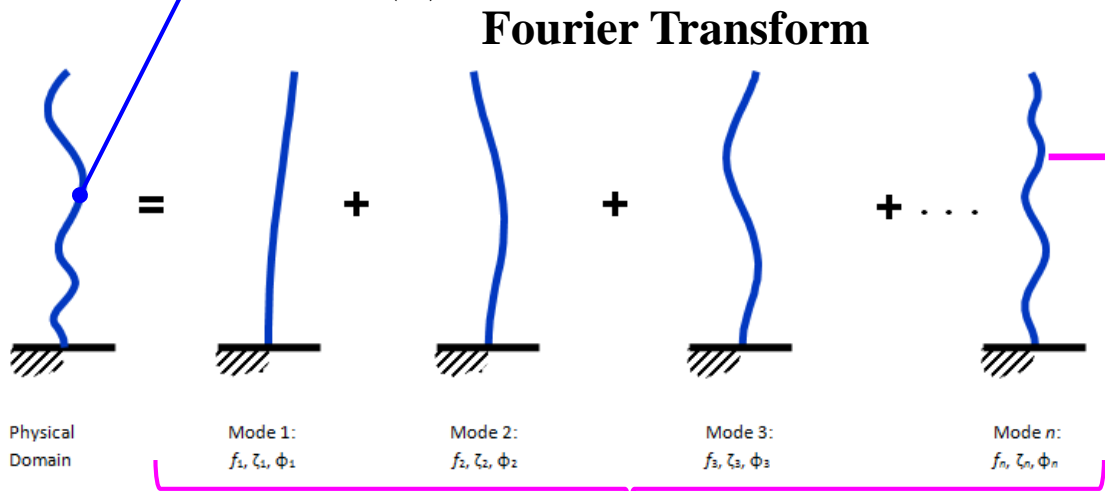
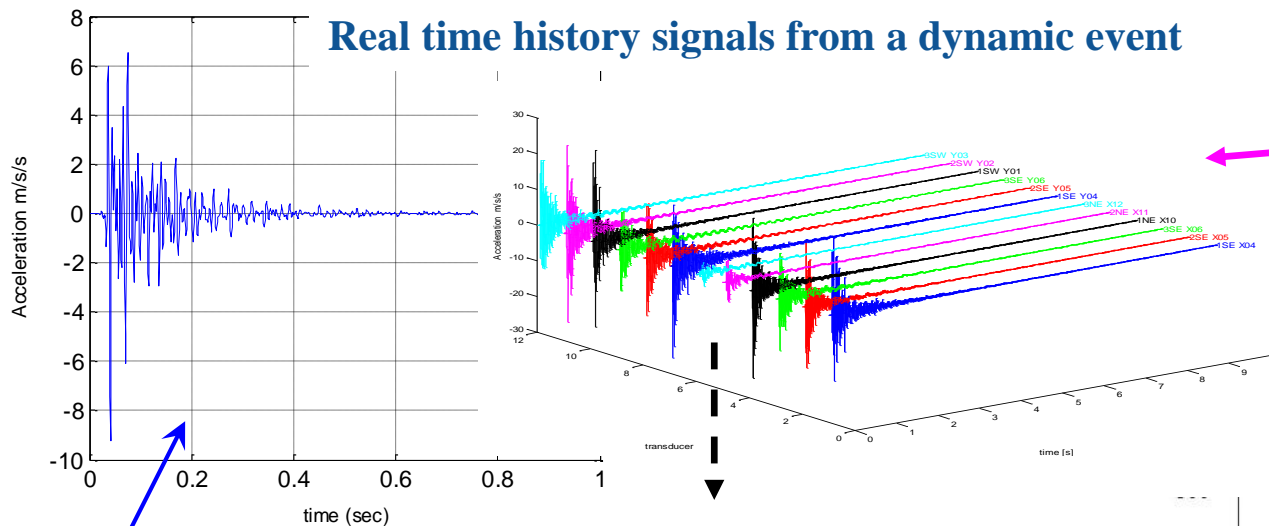
Mode Shapes ϕ_i
Amplitude at different positions j

J positions

- Could give:
- The mechanical state of the structure (damaged or not)
 - Reference values to implement numerical models



Practically



Aim of the Experimental Modal Analysis Toolbox: extract parameters from “complex” signals

The “Genome” of the structural motion



Instrumentation

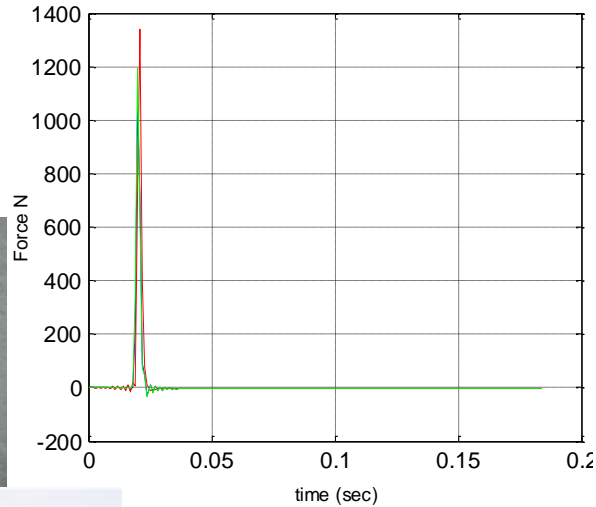
(Input)



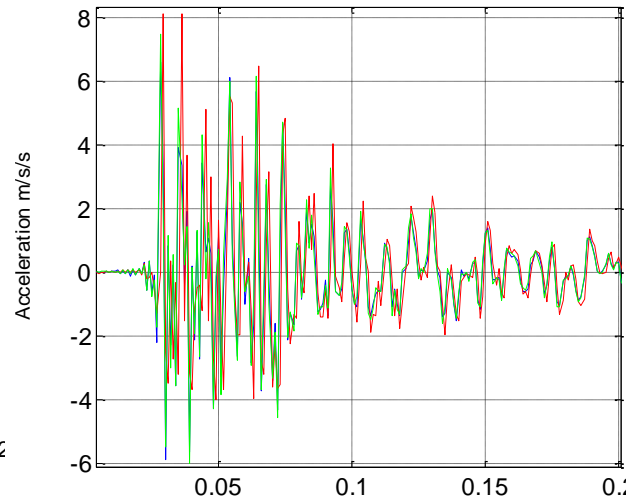
Hammer Impact



Input signals $X(t)$ in position i



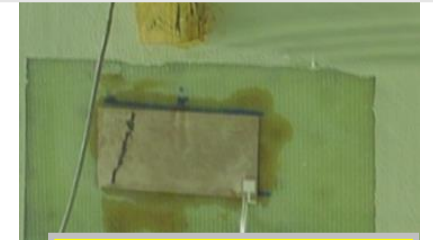
Output signals $Y(t)$ in position j



(Output)



Accelerometer



Piezo-Patch (Embedded sensor)



Displacement transducers



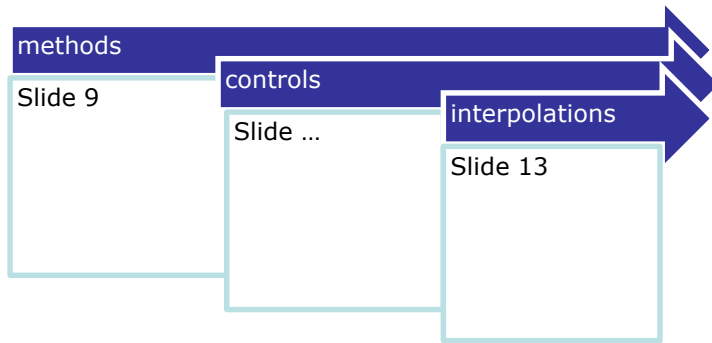
Wireless Acc.



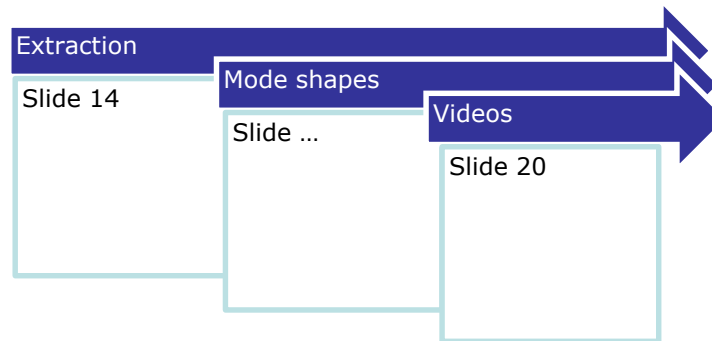
Truck Impact



The toolbox is divided in two parts: in each of them MATLAB eases and shortens the signal processing



1
Process the data during the acquisition



2
After the acquisition extracts the experimental modal parameters displaying results in different forms

Acquisition Methods



The toolbox runs with two types of tests

Fast Impact Hammer Test (FIHT)

Possible only with a fast check of the coherence function (values near 1)

MATLAB 1

j limited by the number of Output transducers

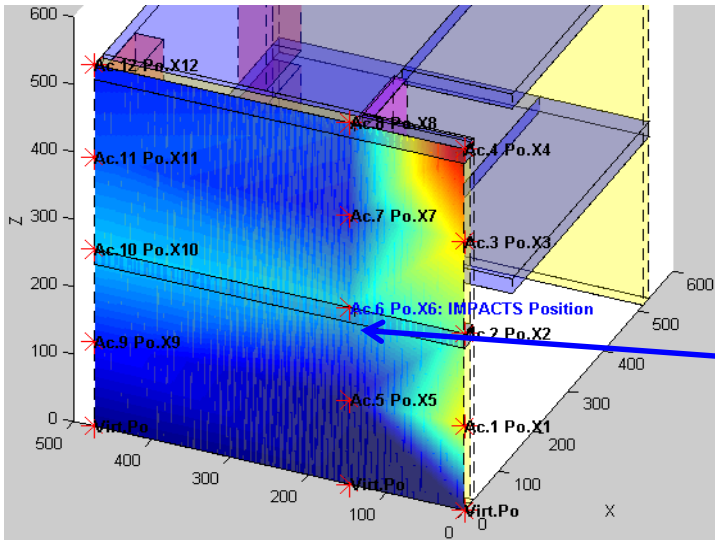
j is not limited ! More experimental data more accurate results. It is possible to test additional positions etc...

Use of reciprocity property of the FRF (Transfer Function)

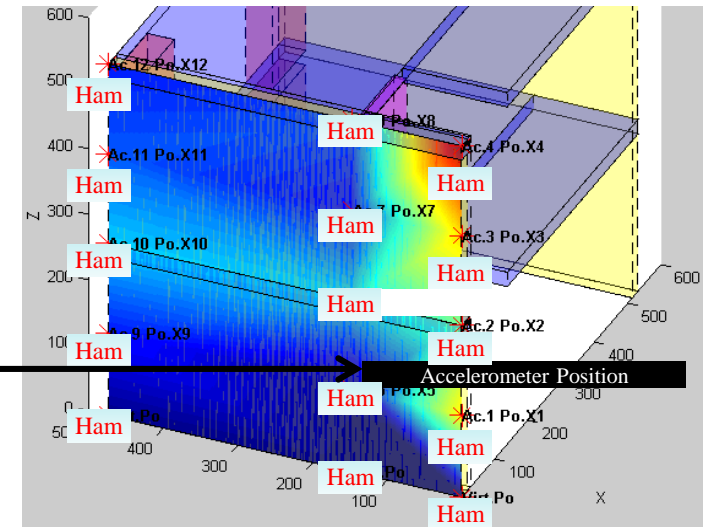
$$H_{ij}(\omega_k) = H_{ji}(\omega_k)$$

$$\frac{A_j(\omega_k)}{F_i(\omega_k)} \stackrel{\text{Ref.}}{=} \frac{A_i(\omega_k)}{F_j(\omega_k)}$$

Equality Limitations



j Accelerometer positions



j Hammer Impact positions

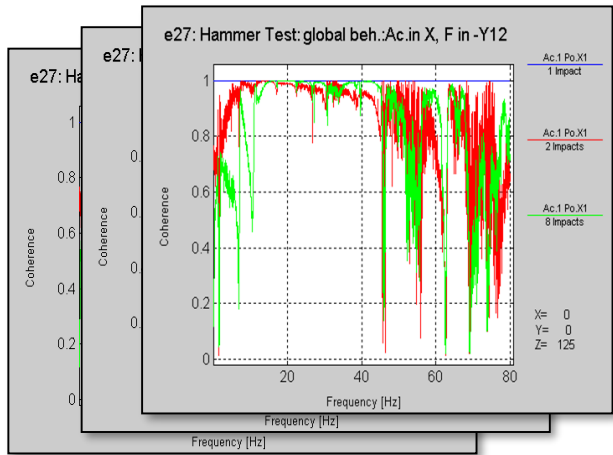
Control of the measurements

Tenths or hundreds of coherence curves to check

A first "Time Reduction" converting number of curves in few histograms

MATLAB: 1

Coherence curves

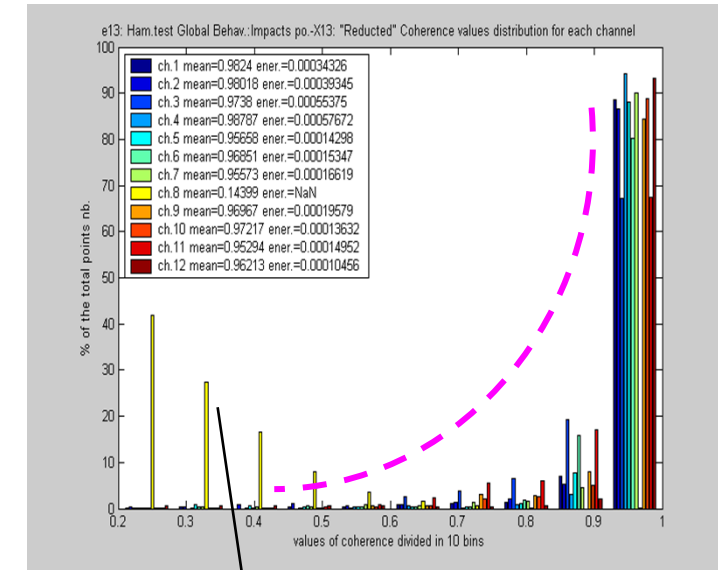


Correct measurement

Represented by histogram of the coherence

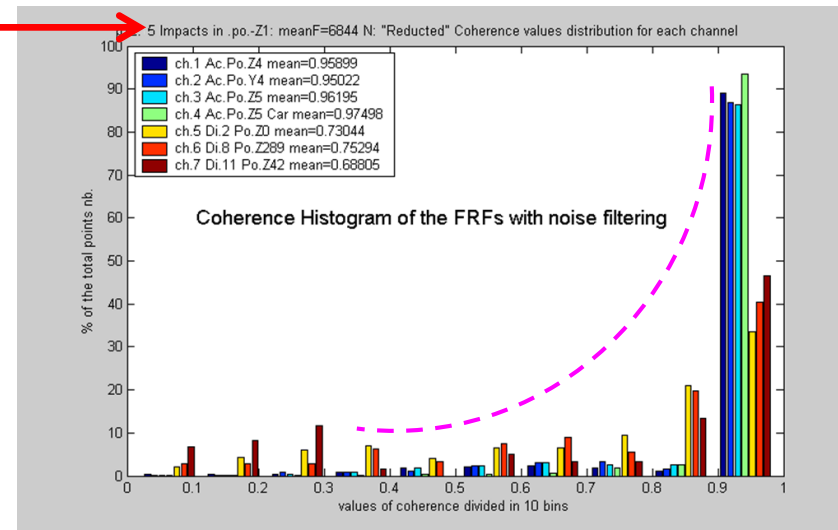
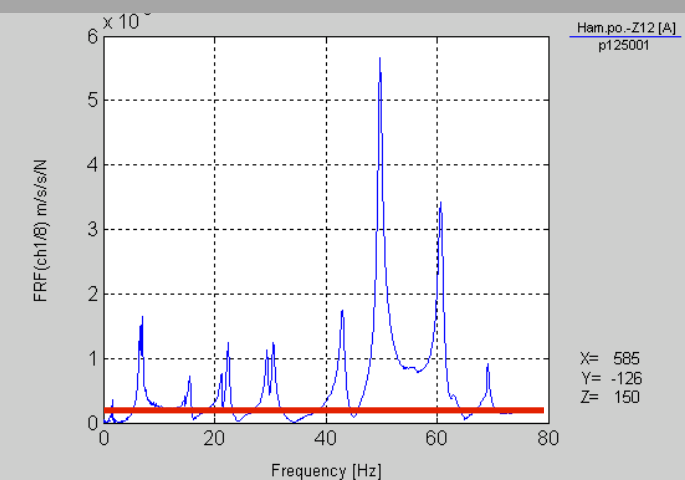
with a

positive exponential fitting



After Noise filtering

FRF curves



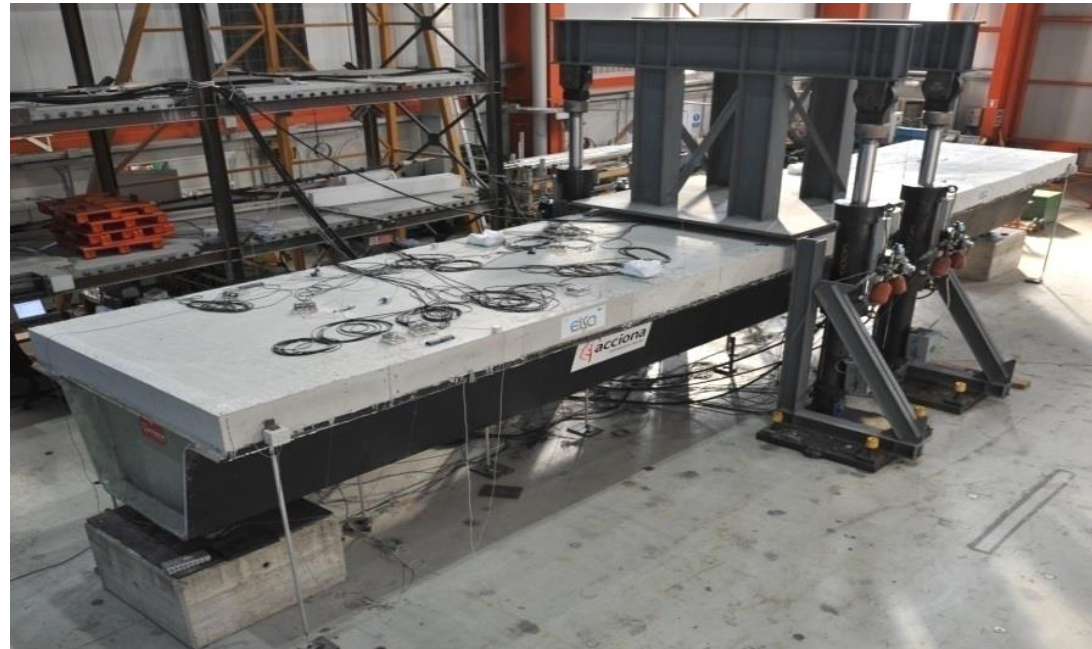
Erroneous measurements ?
(on displacement transducer)



Spatial control of the measurements

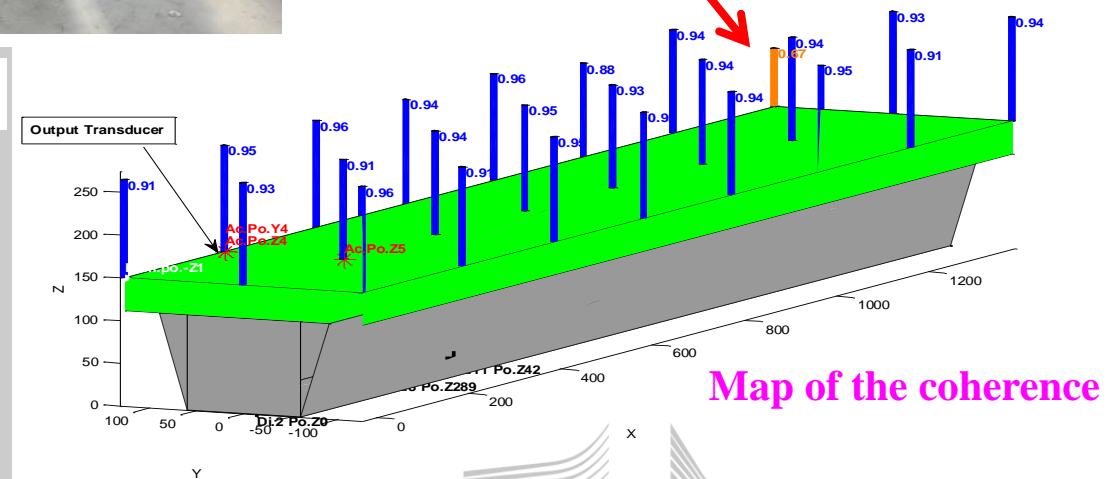
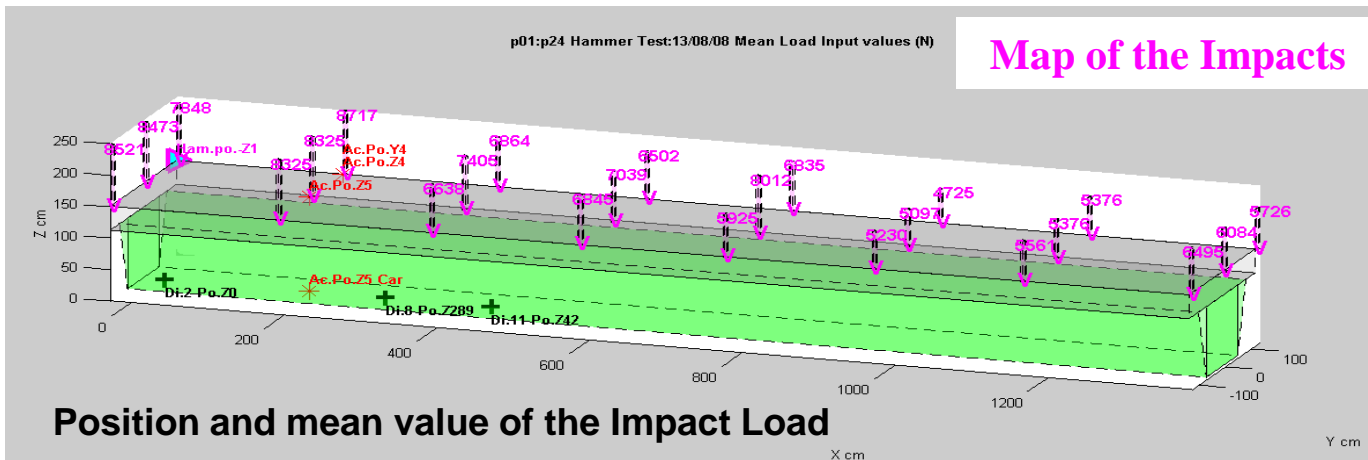
← MATLAB: 2

A Typical Example



Hammer tests on the first composite bridge of a motorway in Europe (Asturia -Spain)

Position to be controlled !

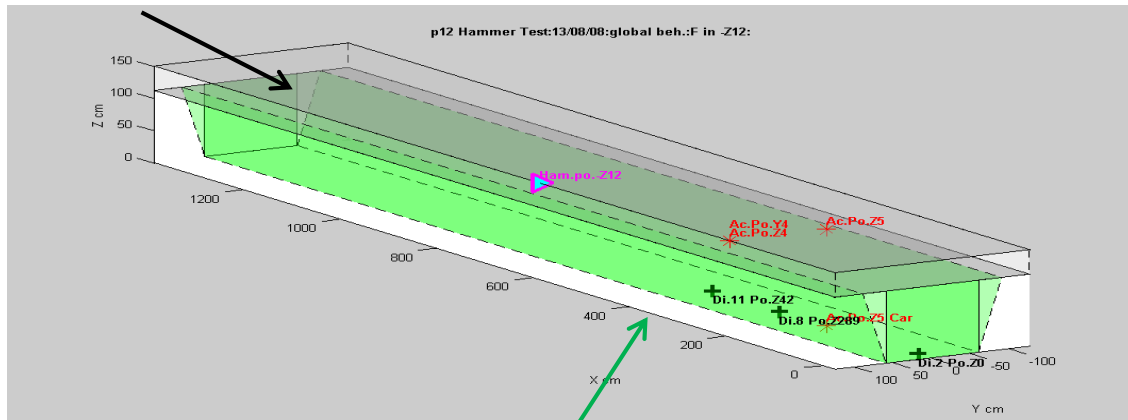


Examples:

Bridge element in ELSA (*Prometeo project*)

Deck designed in grey color :

```
beam([-28 1385],[-126 126],[112 150],nf, [.6 .6 .6]);
```

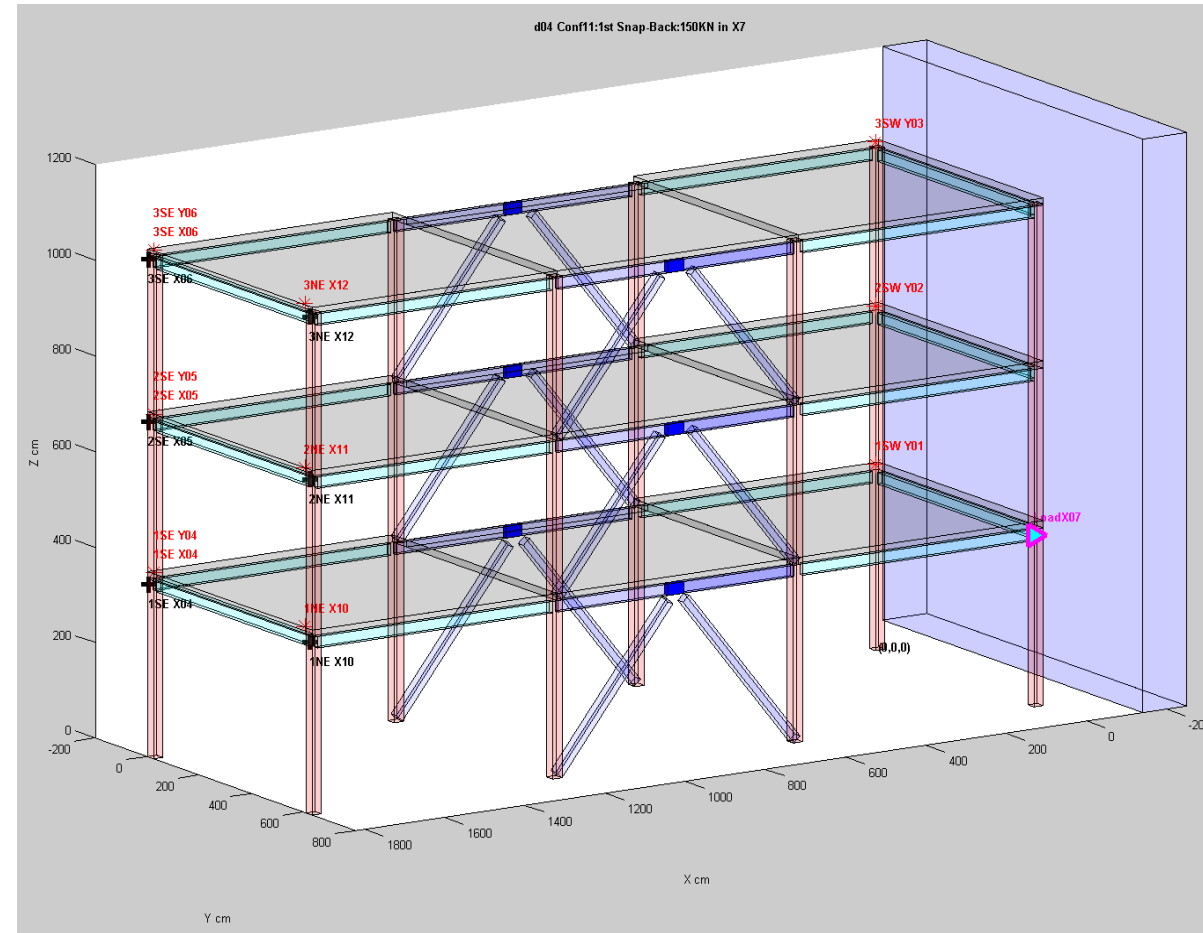


Carbon Beam with irregular section designed in green

```
irsbeam([-28 1385],[60 -60 -90 90 60],[0 0 112 112 0],12,'g')
```

([X], [Y], [Z], fig.Nb., color)

Design of a Building in ELSA (*Duarem project*)





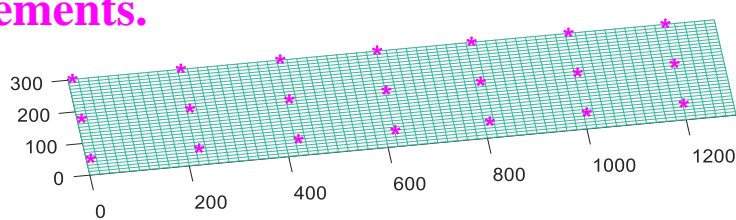
Interpolations of irregular mesh of measurement positions

Color grid in correspondence with the vibration amplitude levels

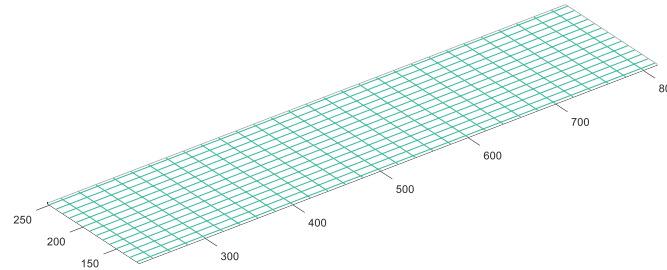
1- Create a regular cartesian mesh:

```
[xi,yi] = meshgrid(mi1:step1:ma1,mi2:step2:ma2)
```

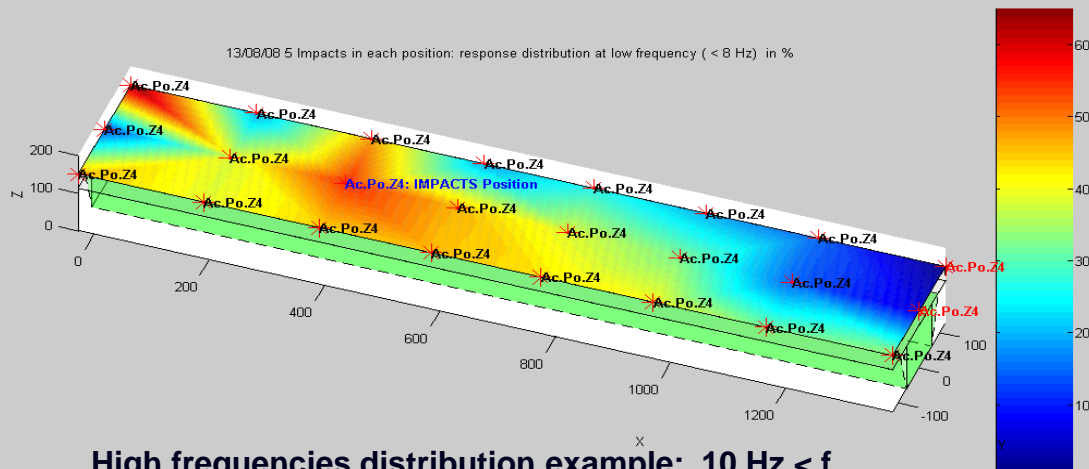
2-Interpolate the grid by the irregular distribution of measurements.



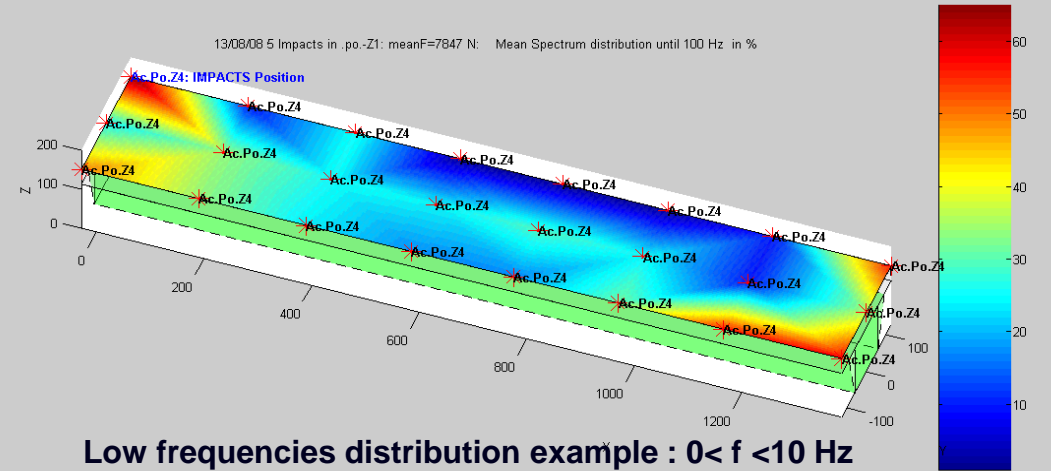
Ex: Prometeo Bridge Project



Produce an iso-surface with: $ci = griddata(Xcap1,Ycap2,Vcou,xi,yi);$



High frequencies distribution example: $10 \text{ Hz} < f$



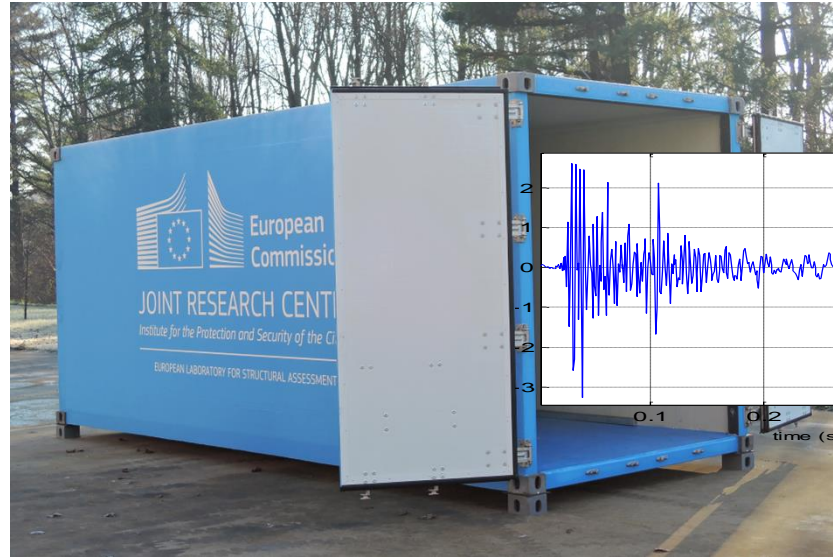
Low frequencies distribution example : $0 < f < 10 \text{ Hz}$



Example of structure to illustrate the process of modal parameters extraction: a smart composite container prototype

Vibrations

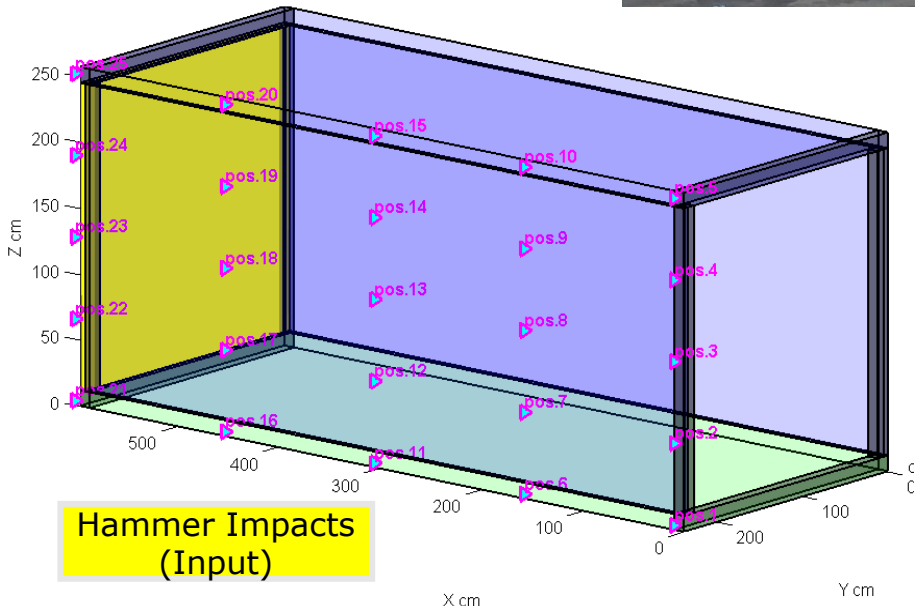
Induced by rain, wind, transportation, damaging etc..



Allows to measure the global mechanical properties of the container or one of its components.



Cstruct Container 1



Hammer Impacts (Input)



Signals in time domain

- 3 impacts at each position
- 25 positions for each panel
- 5 panels (each container)
- 3 transducers (Force+ 2 Acc)
- Total: minimum time signals**

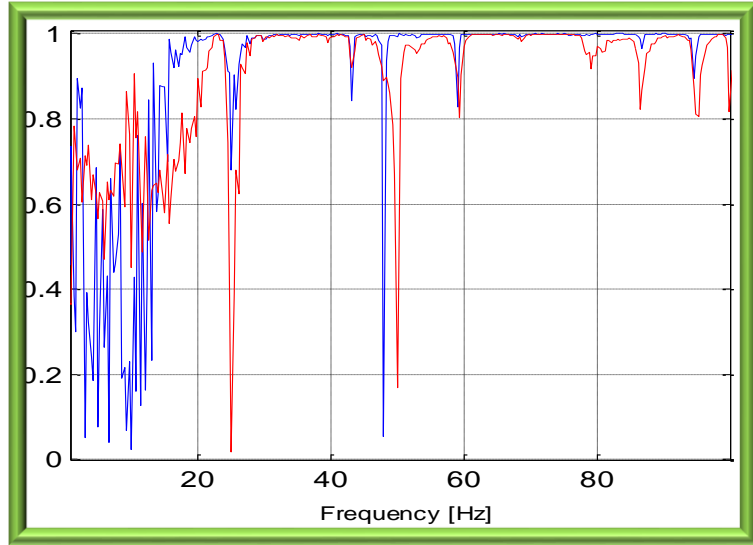
	3
x	25
x	5
x	3
	= 1125

Signal processing to obtain modal parameters

From Time domain (1125 signals) to

Frequency domain (Fourier Transform)

Control of the data



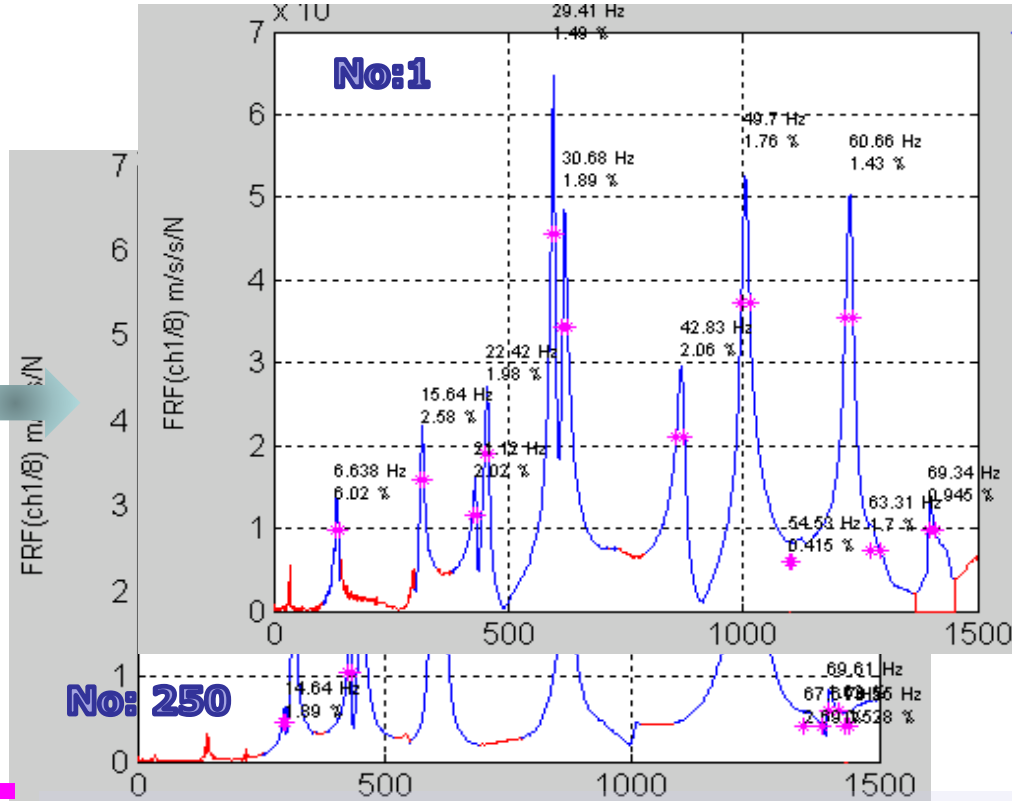
250 Coherences functions
("control" of the measurements)



Matlab, step:5

Algorithm of automatic identification of 3 parameters of each peak in the experimental and noisy 250 curves, and display of the results in a few seconds

Physical interpretation of the data



(125 positions*2 Outputs) =250 FRFs files

Frequency
Amplitude
Damping

3 ×

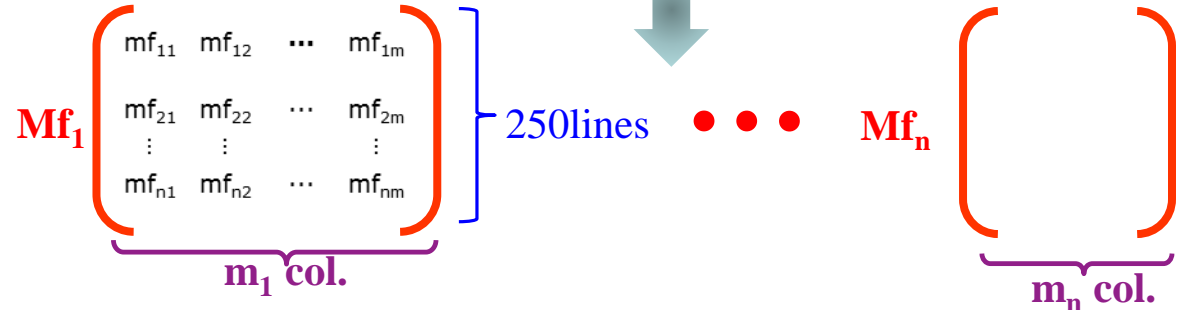
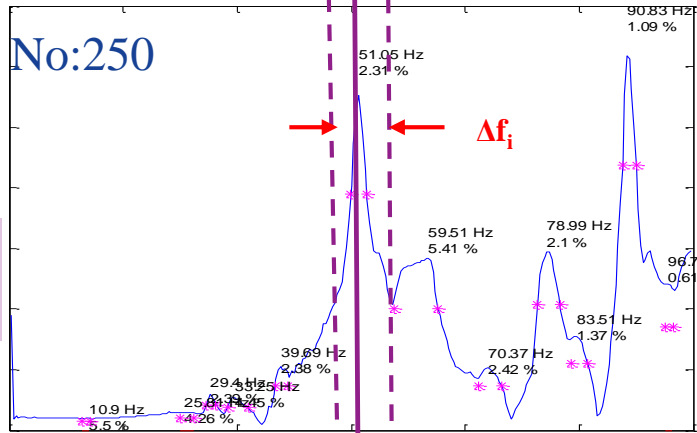
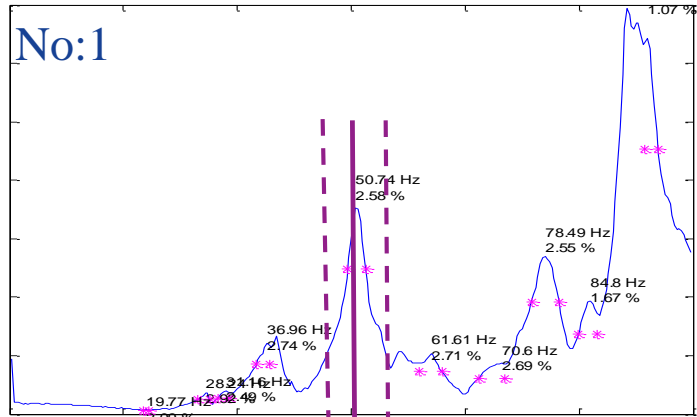
250 vectors
of different
dimensions

Optimisation to obtain the final modal parameters

MATLAB, step:6 (fastening with matrix)

250 frequency vectors

Clustering in m_i modes



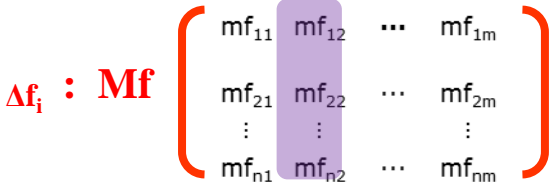
For each window dispersion Δf_i : 1 Frequency Matrix Mf_i

Which is the best clustering?

Calculate for each Δf_i , a Statistic S, and a Physical parameters P:
 S=Mean Std. Deviation for all columns (modes)
 P= Stability of the cluster (modes) when Δf_i increases.

Solution in the set of frequency matrix

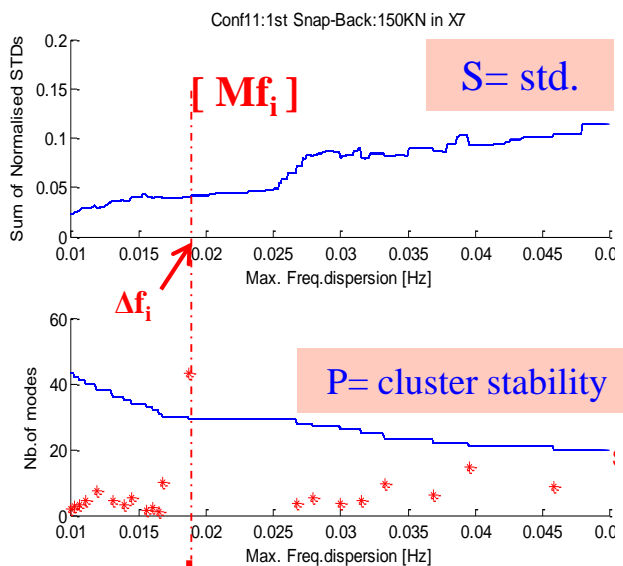
$$= \text{Max}(P/S)$$



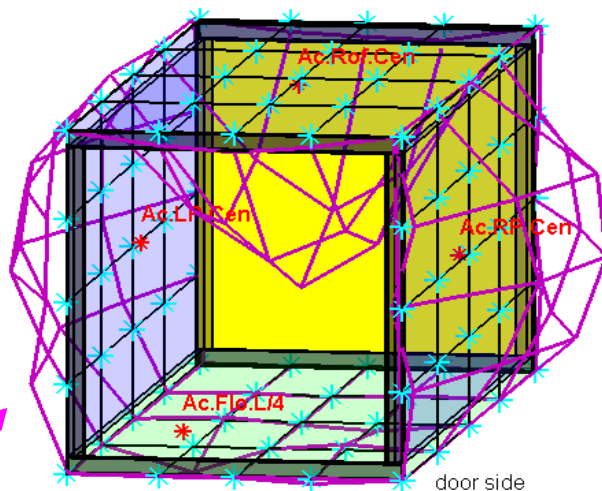
1 column for each frequencies cluster found



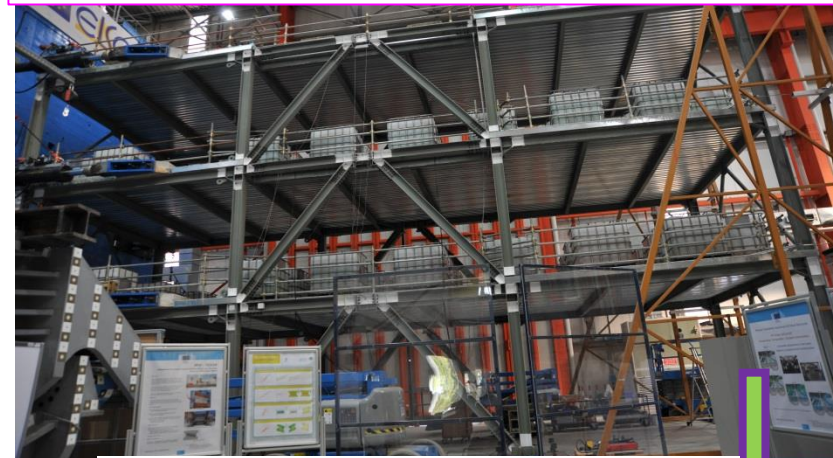
Solution of the modal parameters extraction



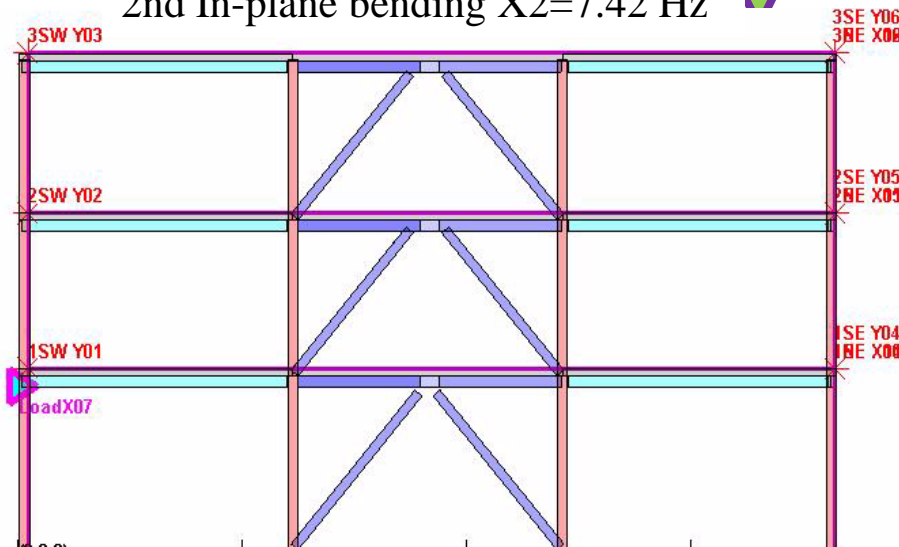
Mode shapes



Other Example: Duarem Frame in ELSA



2nd In-plane bending X2=7.42 Hz



CSTRUCT

25 Highest modes/FRF.
Minimum of 8 values mode
Freq.Dispersion limits=0.3 to 1.3(Hz)
Results from Output in: Acc4,

Solution Number:1 Df<1.0470 Hz				Solution Number:2 Df<0.7830 Hz				Solution Number:3 Df<0.4830 Hz			
Mode Nb.	Freq. (Hz)	Damp. (%)	Fre.Disp. (%)	Mode Nb.	Freq. (Hz)	Damp. (%)	Fre.Disp. (%)	Mode Nb.	Freq. (Hz)	Damp. (%)	Fre.Disp. (%)
01	8.3104	8.052	7.274	01	8.3104	8.052	7.274	01	22.764	1.089	1.422
02	22.764	1.089	1.422	02	22.764	1.089	1.422	02	26.666	3.648	0.801
03	26.791	3.728	3.139	03	26.738	3.537	2.147	03	29.297	1.717	1.325
04	29.280	1.766	3.144	04	29.297	1.717	1.325	04	29.357	1.622	1.185
05	31.301	2.537	2.179	05	31.301	2.537	2.179	05	31.268	2.470	1.125
06	35.294	2.548	2.250	06	42.393	1.480	1.315	06	42.400	1.473	1.175
07	39.343	2.580	1.655	07	44.390	2.096	1.664	07	66.397	1.447	0.633
08	42.393	1.480	1.315	08	66.257	1.433	0.801	08	90.575	1.367	0.418
09	44.491	2.301	1.803								
10	63.185	1.554	1.629								
11	66.269	1.445	1.047								
12	90.515	1.329	0.713								



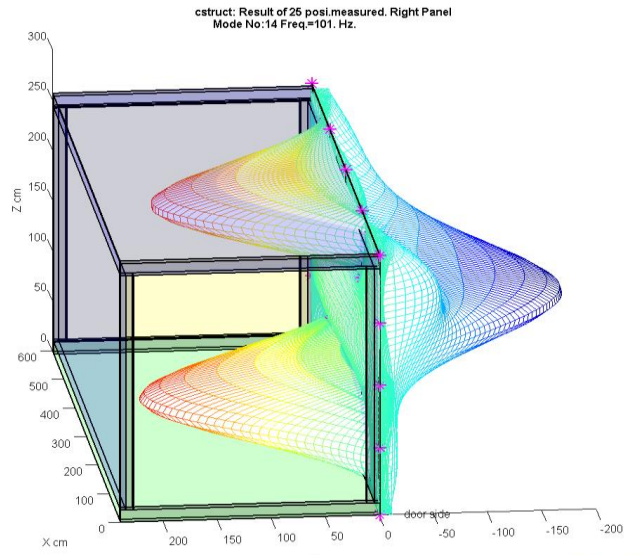
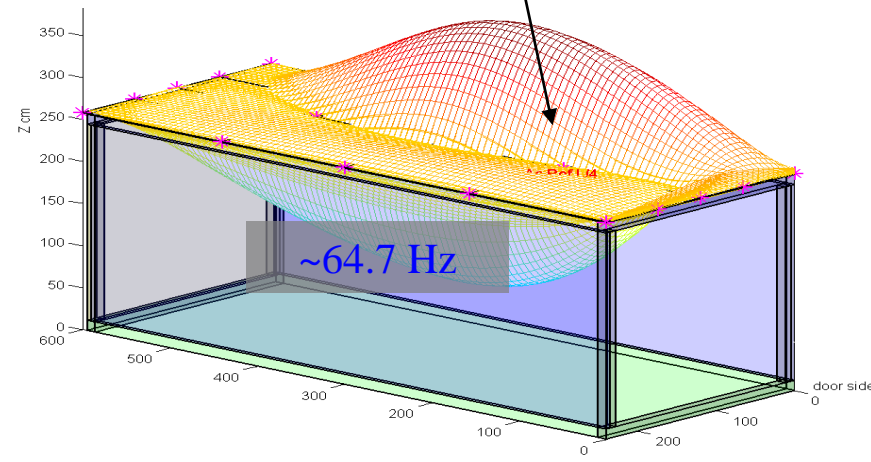
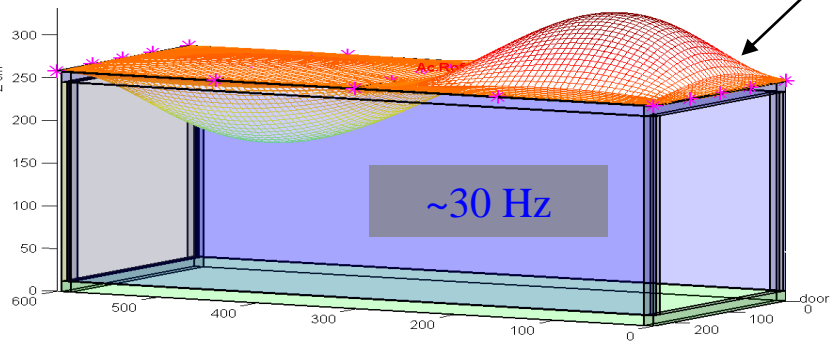
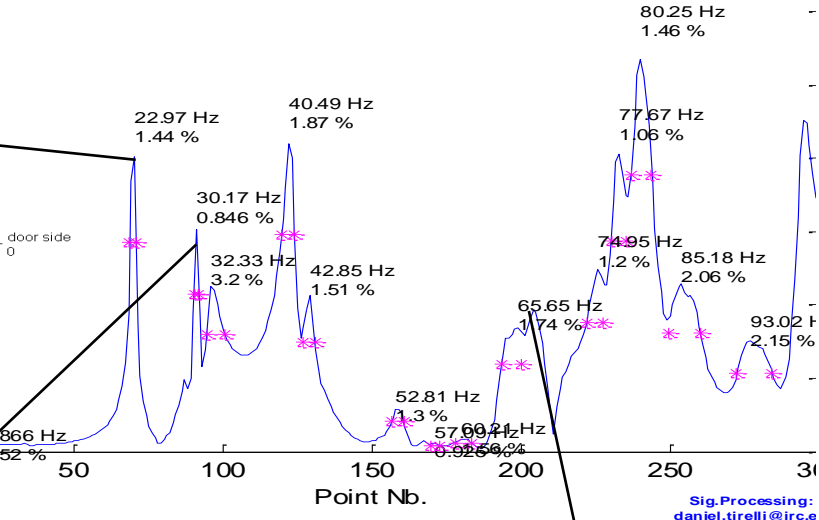
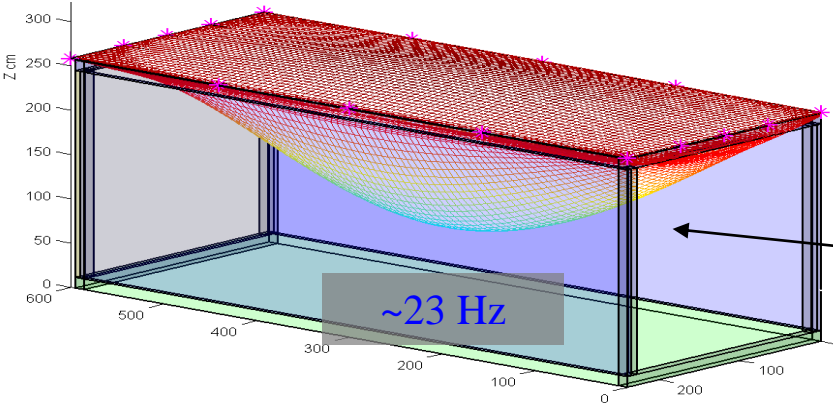
Spline interpolation for a better shape interpretation

Example of the main mode shapes of the roof panel in correspondence of the peaks

MATLAB, step:7



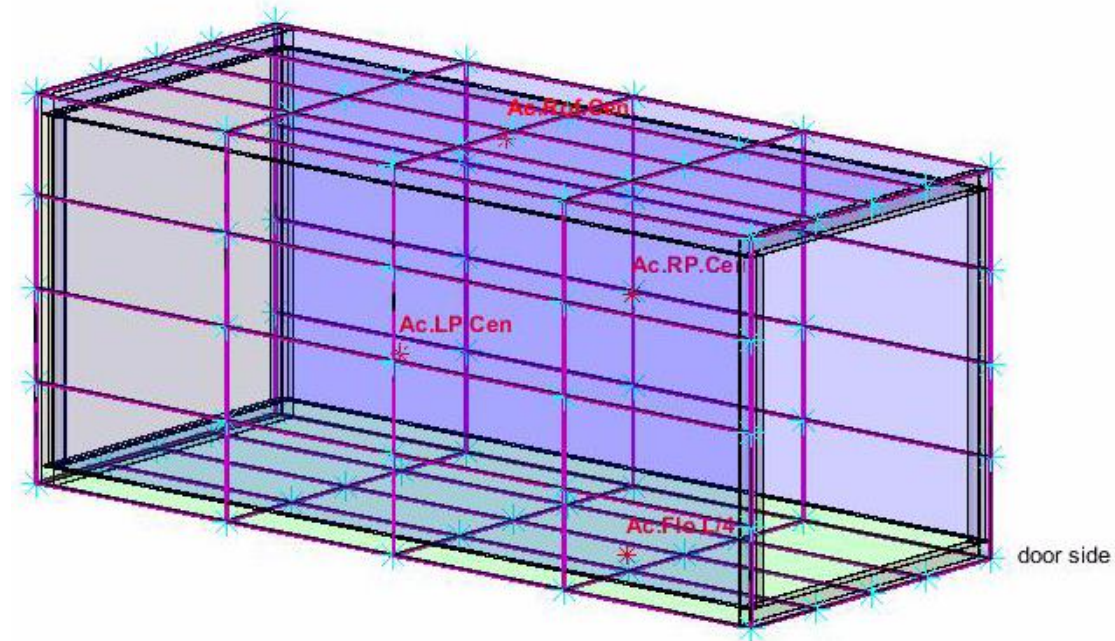
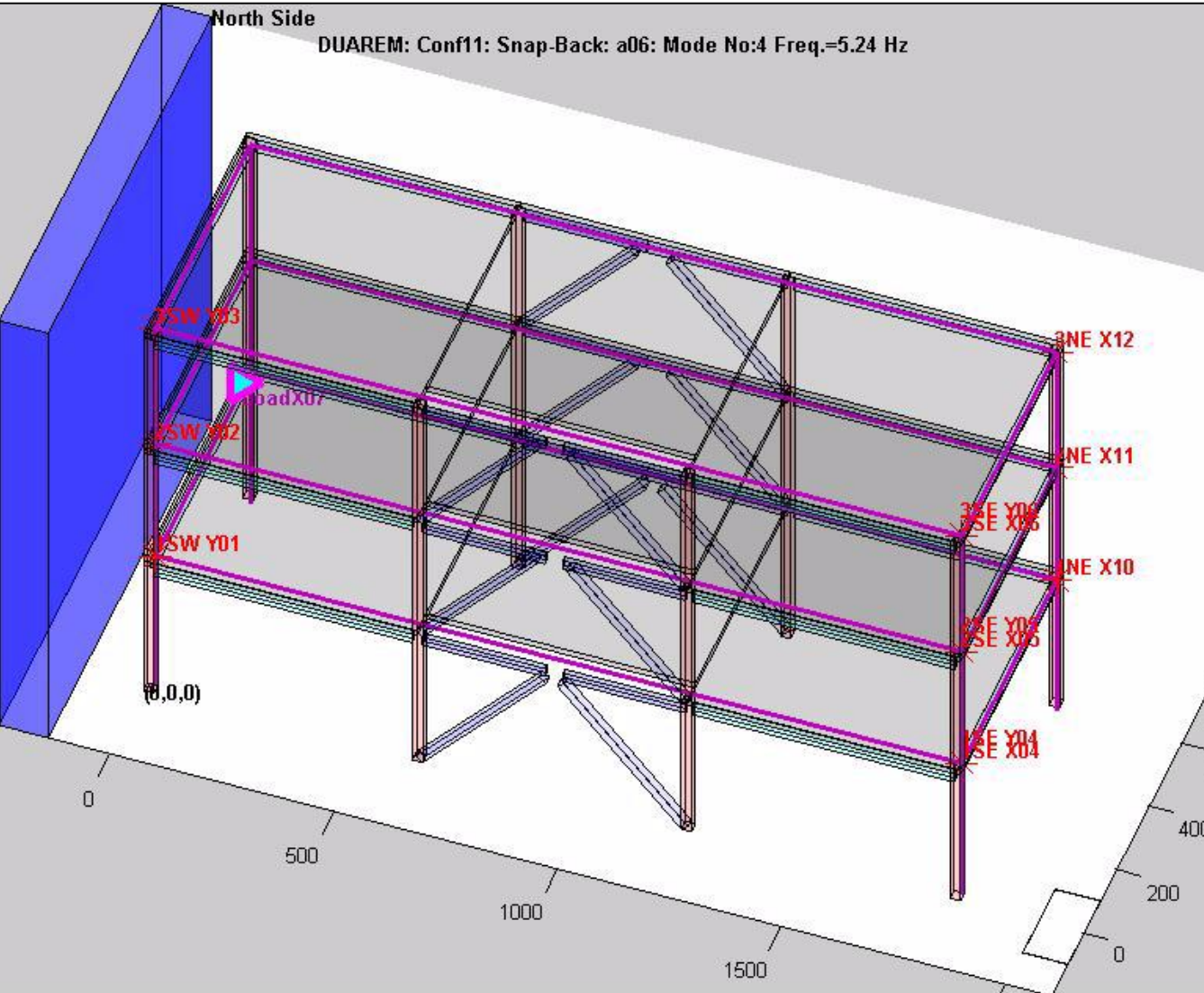
```
[xi,zi] = ndgrid([...],...);
yi = interpn(X',Z',Y',xi',zi','spline');
```



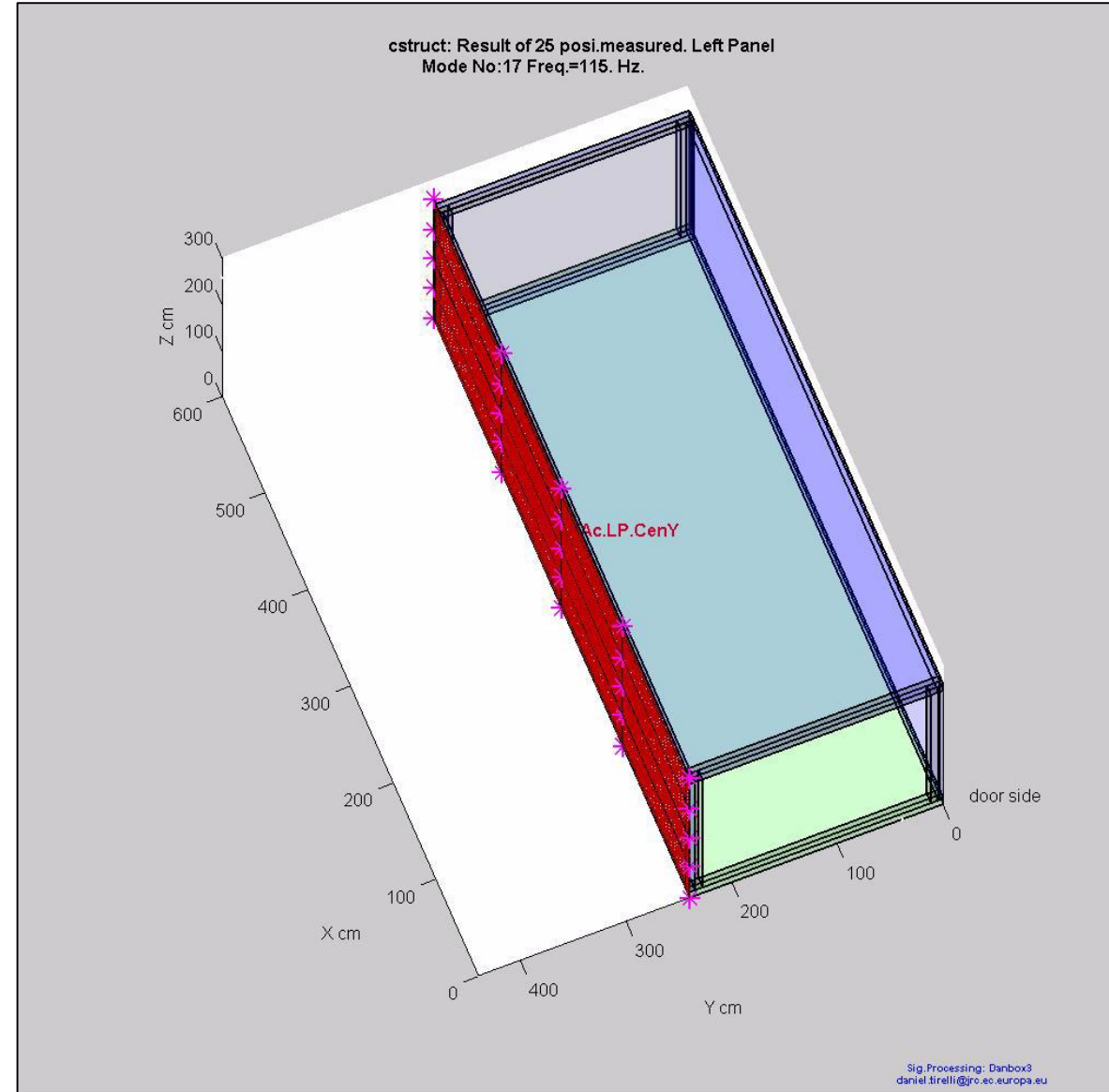
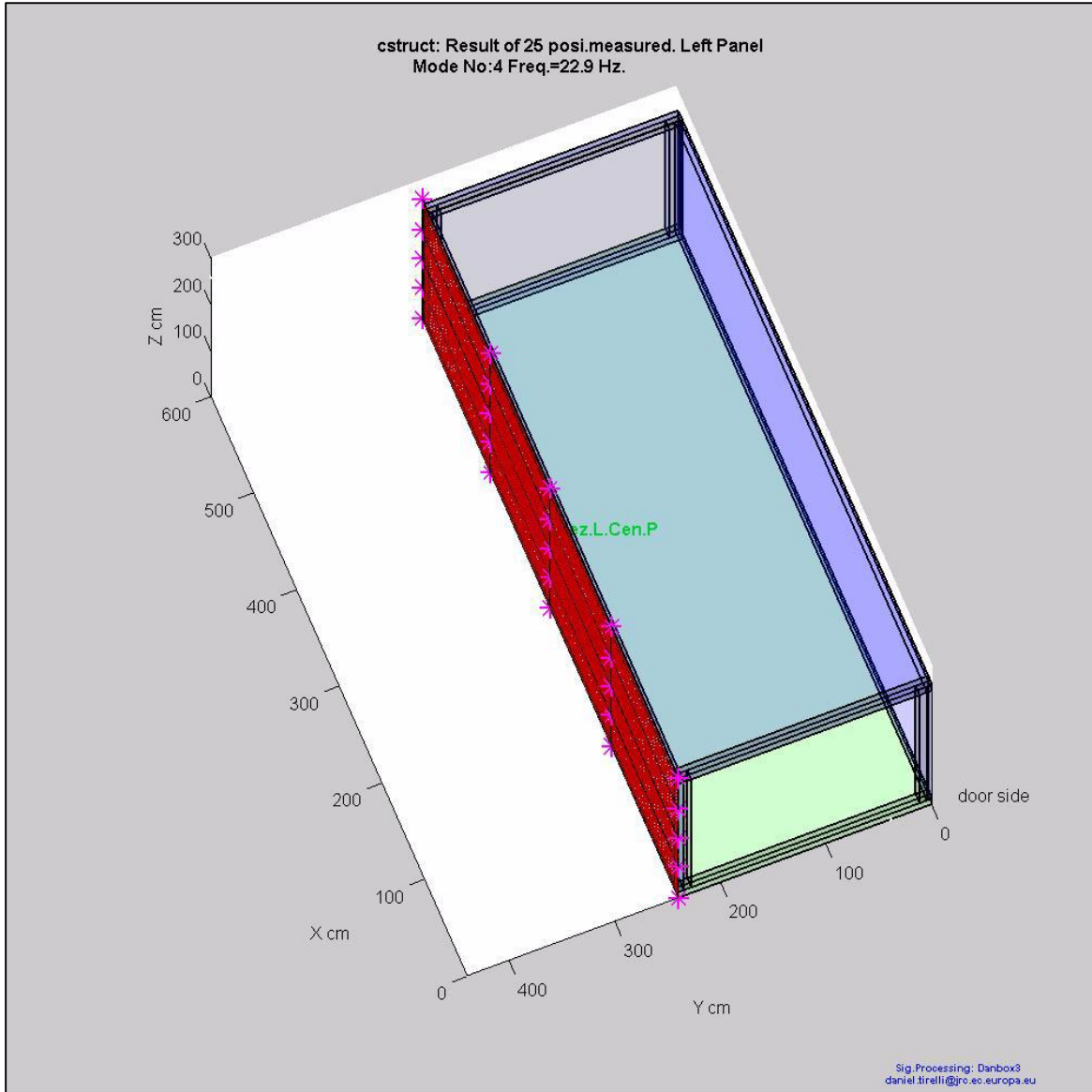
Example of a mode shapes of one of the vertical panel



Results recorded in video format without interpolation



Results recorded in video format with interpolation



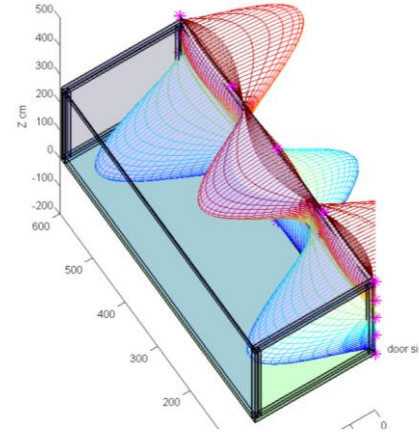
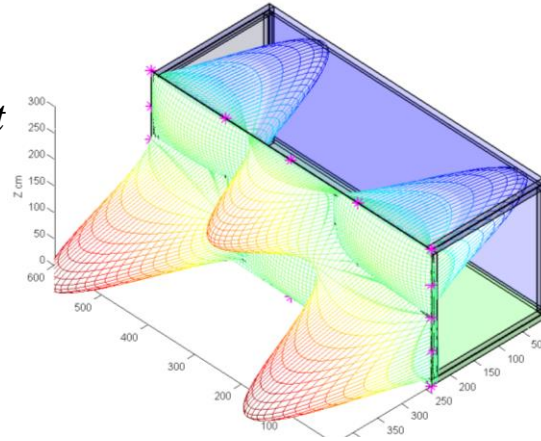


Use and Interpretation of the Results: examples

Stiffness comparison:

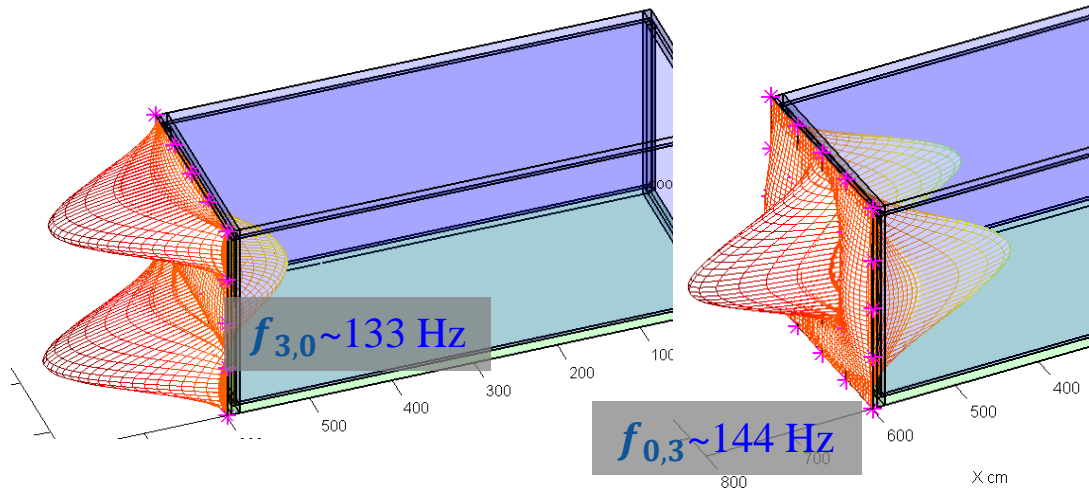
Left panel stiffer than the right panel $f_l > f_r$

Left Panel $f_l \sim 77,2$ Hz



Right panel $f_r \sim 72,3$ Hz

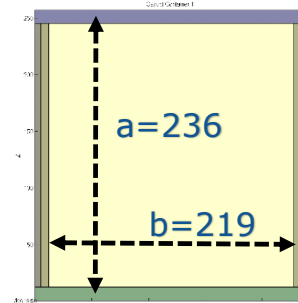
Comparison with theory of thin plates: *in agreement with the theory*



Theory of thin plates

$$f_{m,n} = \frac{c}{2\pi} \sqrt{\left(\frac{m\pi}{a}\right)^2 + \left(\frac{n\pi}{b}\right)^2}$$

Mode ratio	Freq. ratio Exp.	Freq. ratio Theo.
Y2/Z2	1.066	1.077
Y3/Z3	1.08	



Conclusions

The methodology allows the experimental modal analysis based only on measurements processing, with structured automated steps.

Automatisation of:

- **Near Real-time Signal Processing** during the tests for measurements control.
- Possibility to adopt **Fast Hammer Testing** method (faster method)
- **Spatial control** of the measurement to immediately detect the errors locations.
- **Powerful Interpolation** to control and for a great help in the interpretation of the results.
- **3D Visual animated** representation of the results.



Thank you

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European
Commission